

$$A_0 = \begin{bmatrix} 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 & -B_\epsilon & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & -B_\rho & 0 & 0 \\ 0 & U & 0 & 0 & 0 & 0 & -B_\lambda & 0 \\ 0 & 0 & 0 & M_p g l & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$B_0 = \begin{bmatrix} 0 & 0 & 0 & 0 & K_f L_a & K_f L_h & 0 & 0 \\ 0 & 0 & 0 & 0 & K_f L_a & -K_f L_h & 0 & 0 \end{bmatrix}^T$$

Where J_ϵ , J_ρ and J_λ [kg-m²] are each moment of inertia.

2.2 Servo system

We synthesis a controller that follows a provided reference. In order to remove a steady-state error, a servo system is used. The error between the observed output $y(t)$ and the reference $r(t)$ is $e(t)$. $\tilde{w}(t)$ is $[r_\epsilon(t) \ r_\lambda(t)]^T$. The state variable of the servo system satisfy $x(t) = [x_p(t) \ \int_0^t e_\epsilon(t)dt \ \int_0^t e_\lambda(t)dt]^T$. The servo system is expressed as follows.

$$\begin{cases} \dot{x}(t) = Ax(t) + \tilde{B}_1 \tilde{w}(t) + B_2 u(t) \\ \tilde{z}(t) = \tilde{C}_1 x(t) \\ y(t) = C_2 x(t) \end{cases} \quad (2)$$

Where matrices A , \tilde{B}_1 , B_2 , \tilde{C}_1 and C_2 are given as follows.

$$A = \begin{bmatrix} A_0 & 0 \\ -C_e & 0 \end{bmatrix}, \quad C_e = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$\tilde{B}_1 = \begin{bmatrix} 0 \\ I \end{bmatrix}, \quad B_2 = \begin{bmatrix} B_0 \\ 0 \end{bmatrix}, \quad \tilde{C}_1 = \begin{bmatrix} 0 & I \end{bmatrix}$$

$$C_2 = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

2.3 Extended system

We consider to realize stability for the swing angle $\theta(t)$ of the suspended load that is not able to be measured directly. Where $w_\theta(t)$ is disturbance for $\theta(t)$. $w(t)$ is $[w_\theta(t) \ \tilde{w}(t)]$. The extended system is expressed as follows.

$$\begin{cases} \dot{x}(t) = Ax(t) + B_1 w(t) + B_2 u(t) \\ z(t) = C_1 x(t) + D_{12} u(t) \\ y(t) = C_2 x(t) \end{cases} \quad (3)$$

Where matrices B_1 , C_1 and D_{12} are given as follows.

$$B_1 = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}^T$$

$$C_1 = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & W_\theta & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & W_{e\epsilon} & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & W_{e\lambda} \end{bmatrix}$$

$$D_{12} = \begin{bmatrix} W_u \\ 0 \end{bmatrix}$$

Where W_θ , $W_{e\epsilon}$, $W_{e\lambda}$ and W_u are each weighting constants for the states and the input.

3 CONTROLLER SYNTHESIS

3.1 Robust stabilization problem

Mass of the suspended load may not be the same all the time. Where $\Delta_m(s)$ is uncertainty of the multiplication of the nominal plant and the perturbed plant. We apply a small gain theorem and derive a frequency weight W_t satisfying $\bar{\sigma}\{\Delta_m(j\omega)\} < |W_t(j\omega)|$. Where matrix W_t is given as follows.

$$W_t = \begin{bmatrix} A_t & B_t \\ C_t & D_t \end{bmatrix}$$

Singular plots of the frequency weight and uncertainty of the multiplication are shown in Figure2. Now, the range of the varying parameter M_p is $2.85 \leq M_p \leq 3.15$. We choose the frequency weight to cover each uncertainty of the multiplication. In Figure2, a dotted line is the frequency weight W_t . The generalized plant $G(s)$ is expressed as follows.

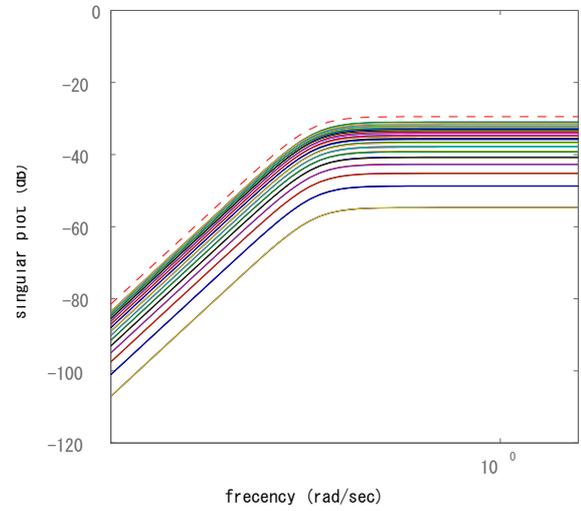


Figure 2 $\bar{\sigma}\{\Delta_m(j\omega)\} < |W_t(j\omega)|$

$$G(s) = \left[\begin{array}{cc|cc} A & 0 & B_1 & B_2 \\ B_1 C_3 & A_t & 0 & 0 \\ \hline C_1 & 0 & D_{11} & D_{12} \\ D_t C_3 & C_t & 0 & 0 \\ C_2 & 0 & D_{21} & 0 \end{array} \right] \quad (4)$$

Where matrix C_3 is given as follows.

$$C_3 = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

3.2 H_∞ controller synthesis using output feedback

Now, Eq(4) is expressed as Eq(5). We design the output feedback controller K expressed as Eq(6) using H_∞ controller synthesis for the generalized plant Eq(5).

$$G : \begin{cases} \dot{x}(t) = A_G x(t) + B_{G1} w(t) + B_{G2} u(t) \\ z(t) = C_{G1} x(t) + D_{G11} w(t) + D_{G12} u(t) \\ y(t) = C_{G2} x(t) + D_{G21} w(t) \end{cases} \quad (5)$$

$$K : \begin{cases} \dot{x}_K(t) = A_K x_K(t) + B_K y(t) \\ u(t) = C_K x_K(t) + D_K y(t) \end{cases} \quad (6)$$

There is a constant value $\gamma_\infty > 0$. If $X \in \mathbb{S}_{++}^n$, $Y \in \mathbb{S}_{++}^n$, $\hat{A}_K \in \mathbb{R}^{n \times n}$, $\hat{B}_K \in \mathbb{R}^{n \times n_y}$, $\hat{C}_K \in \mathbb{R}^{n_u \times n}$ and $\hat{D}_K \in \mathbb{R}^{n_u \times n_y}$ which satisfy following LMIs(7), (8) exist, the closed loop system is stable. Also the output feedback control gain which realize $\|G\|_\infty < \gamma_\infty$ is given as Eq(6).

$$\begin{bmatrix} X & I \\ I & Y \end{bmatrix} > 0 \quad (7)$$

$$\begin{bmatrix} He(A_G X + B_{G2}) \hat{C}_K & * \\ \hat{A}_K + (A_G + B_{G2} \hat{D}_K C_{G2})^T & He(YA + \hat{B}_K C_{G2}) \\ B_{G1}^T + D_{G21}^T \hat{D}_K^T B_{G2}^T & B_{G1}^T Y + D_{G21}^T \hat{B}_K^T \\ C_{G1} X + D_{G12} \hat{C}_K & C_{G1} + D_{G12} \hat{D}_K C_{G2} \\ * & * \\ * & * \\ -\gamma_\infty^2 I & * \\ D_{G11} + D_{G12} \hat{D}_K D_{G21} & -I \end{bmatrix} < 0 \quad (8)$$

We derive desired H_∞ controller using $X, Y, \hat{A}_K, \hat{B}_K, \hat{C}_K$ and \hat{D}_K satisfying the LMIs(7), (8). Where matrices $I - XY$ and $M, N \in \mathbb{R}^{n \times n}$ are regular. They satisfy $I - XY = MN^T$.

$$\left. \begin{aligned} D_K &:= \hat{D}_K \\ C_K &:= (\hat{C}_K - D_K C_{G2} X) M^{-T} \\ B_K &:= N^{-1} (\hat{B}_K - Y B_{G2} D_K) \\ A_K &:= N^{-1} (\hat{A}_K - N B_K C_{G2} X - Y B_{G2} C_K M^T \\ &\quad - Y (A_G + B_{G2} D_K C_{G2}) X) M^{-T} \end{aligned} \right\} \quad (9)$$

4 SIMULATION

The output feedback gain is obtained by solving the LMIs(7), (8). Then, $D_{11} = 0, D_{21} = 0$, and matrices M and N is derived by LU decomposition.

4.1 Simulation adding disturbance

The weighting constants $W_{e\epsilon}, W_{e\lambda}$ and W_u are chosen by trial and error. We set the reference of $\epsilon(t)$ [deg] and $\lambda(t)$ [deg] to 20[deg] and 200[deg] respectively. Then, we add the wind disturbance for the suspended load at 30[sec] to the simulation. The simulation results are shown in Figure3, Figure4, Figure5 and Figure6.

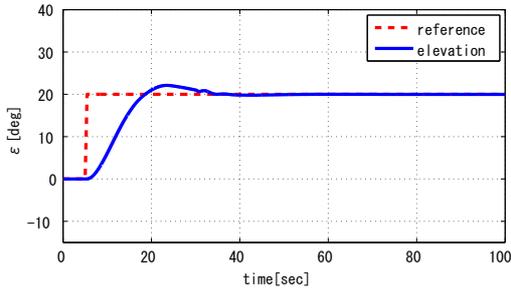


Figure 3 elevation(adding disturbance)

In Figure3 and Figure4, it can be seen that results of the simulation of the elevation and the traveling converge in the reference. Thus, we were able to design the controller for position control of the 3-DOF helicopter. In Figure5, it can be seen that the simulation of the swing angle of the suspended load suppresses the wind disturbance at 30[sec]. Thus, we were able to design the controller for stabilization of the state that is not able to be measured directly by output feedback.

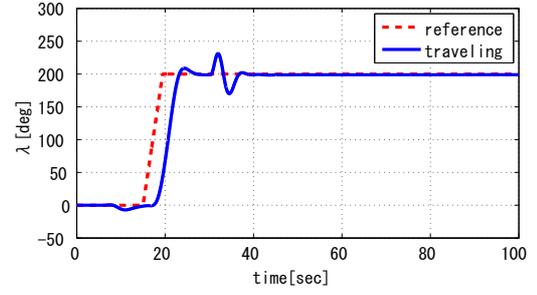


Figure 4 traveling(adding disturbance)

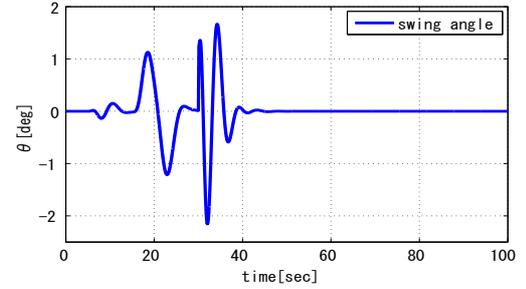


Figure 5 swing angle(adding disturbance)

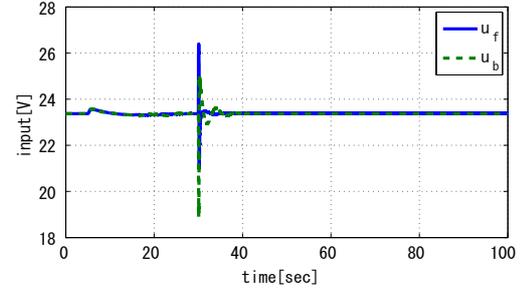


Figure 6 input(adding disturbance)

4.2 Simulation using nominal controller

We simulate to verify the robust stability. At first, we use the controller of not considering the robust stability. Then, mass of the suspended load M_p is 0.15[kg] heavier than the nominal load. The simulation results are shown in Figure7, Figure8, Figure9 and Figure10.

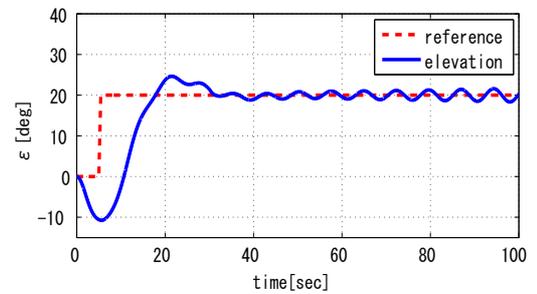


Figure 7 elevation(nominal controller)

In Figure7, Figure8, Figure9 and Figure10, the signals are not able to be stabilized when we used the nominal controller.

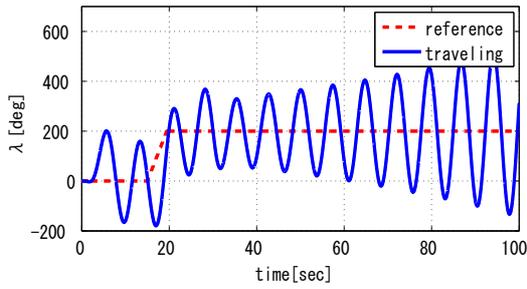


Figure 8 traveling(nominal controller)

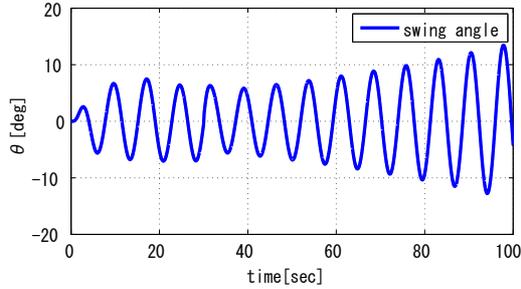


Figure 9 swing angle(nominal controller)

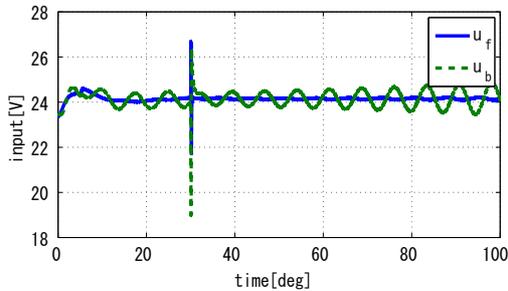


Figure 10 input(nominal controller)

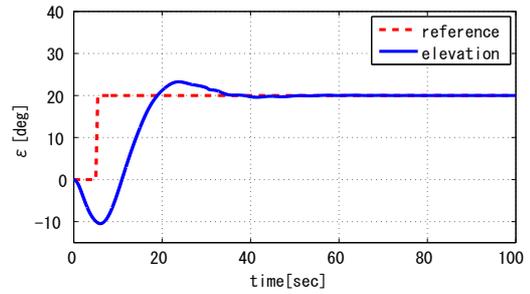


Figure 11 elevation(robust controller)

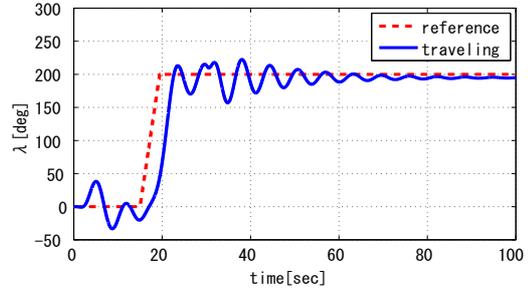


Figure 12 traveling(robust controller)

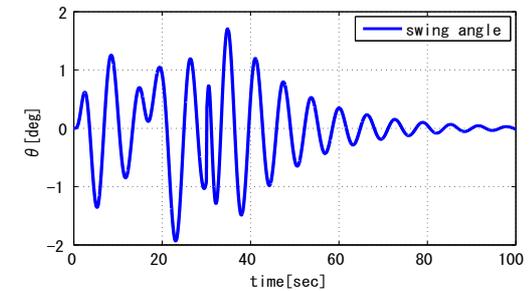


Figure 13 swing angle(robust controller)

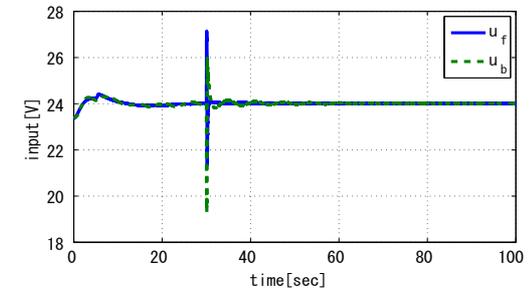


Figure 14 input(robust controller)

4.3 Simulation using robust controller

Next, we use the controller which guaranteed the robust stability on a condition same as 4.2. The simulation results are shown in Figure11, Figure12, Figure13 and Figure14.

In Figure11, Figure12, Figure13 and Figure14, the signals are able to be stabilized when we used the proposed controller. Thus, the controller which is guaranteed the robust stability for varying parameter by H_∞ control using output feedback is effective.

5 CONCLUSION

In this paper, we show the stabilization for the unmeasurable state using output feedback. In addition, we realized robust stability by H_∞ control using output feedback for the control object which has a varying parameter including the state space representation.

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